



PROGRAM : NATIONAL DIPLOMA
ENGINEERING: MECHANICAL TECHNOLOGY

SUBJECT : **THERMODYNAMICS 3**

CODE : **IMT313**

DATE : MID YEAR MAIN EXAMINATION
13 JUNE 2014

DURATION : 08:30 - 11:30

WEIGHT : 40:60

FULL MARKS : 100

TOTAL MARKS : 105

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NUMBER OF PAGES : 5 QUESTIONS ON 3 PAGES AND 2 ANNEXURES
(8 PAGES TOTAL)

INSTRUCTIONS:

- ONE CALCULATOR OF ANY MAKE OR MODEL IS PERMITTED.

REQUIREMENTS:

- NONE

2/

INSTRUCTIONS TO STUDENTS:

- ANSWER ALL QUESTIONS.
 - MAKE REASONABLE ASSUMPTIONS FOR DATA NOT SUPPLIED.
 - NUMBER QUESTIONS CLEARLY UNDERLINING FINAL ANSWERS.
 - ANSWERS WITHOUT UNITS WILL NOT BE MARKED.
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QUESTION 1

The pressure and temperature at the start of an Otto Cycle are 101,3 kPa and 27 °C respectively. The compression ratio during the cycle is 5,7. The heat added during the cycle is 8,7 kJ. The mass of air processed during the cycle is 0.051kg. The air constants that can be assumed are $R = 287 \text{ J/kgK}$; $\gamma = 1,4$; $C_p = 1005 \text{ J/kgK}$ and $C_v = 718 \text{ J/kgK}$.

- 1.1 Draw a neatly labeled and proportioned PV and TS diagram for the cycle. (4)

Calculate the:

- 1.2 Pressure, volume and temperature values at each state point on the cycle. (Tabulate your answers with units) (11)
- 1.3 Positive work output from the cycle. (2)
- 1.4 Negative work input to the cycle. (2)
- 1.5 Net work done during the cycle. (1)
- 1.6 Heat rejected during the cycle. (1)
- 1.7 Thermal efficiency of the cycle. (2)
- 1.8 Work ratio of the cycle. (2)

[25]

QUESTION 2

A two stage single acting reciprocating compressor takes in air at 92 kPa and 21 °C and compresses it to a delivery pressure of 530 kPa. The index of compression and expansion for both stages is 1,2. The clearance volume of the low pressure stage is 3,8% of its swept volume. The volume of air delivered at free air conditions is 8,6 m³ per minute. Free air conditions can be taken to be 101,3 kPa and 15 °C. The compressor is operating at minimum work conditions, ideal intermediate pressure and perfect intercooling.

- 2.1 Draw a neat, fully labeled and proportioned PV diagram for the cycle. (5)
- 2.2 Calculate the ideal intermediate pressure. (2)
- 2.3 Calculate the clearance volume, swept volume and the effective swept volume for the low pressure stage in m³ per second. (8)
- 2.4 Calculate the power required to drive the compressor. (2)
- 2.5 Calculate low pressure stage volumetric efficiency and clearance ratio. (4)
- 2.6 Calculate the low pressure stage isothermal efficiency. (4)

[25]

QUESTION 3

A commercial refrigerator making use of Refrigerant 22 operates between a maximum and minimum pressure of 7,8915 bar and 2,0098 bar respectively. The refrigerator consists of a compressor, condenser, throttle valve and evaporator. The refrigerant is compressed isentropically in the compressor to 20K above saturation temperature. At exit from the condenser, the refrigerant is a saturated liquid with no undercooling.

Properties of Refrigerant 22 are listed below:

t (°C)	P _s (bar)	v _g (m ³ /kg)	10K		20K	
			h _f (kJ/kg)	S _f (kJ/kgK)	h (kJ/kg)	S (kJ/kgK)
-25	2,0098	0,11119	171,61	0,8919	401,74	1,8187
15	7,8915	0,02999	217,94	1,0631	418,26	1,7578

- 3.1 Draw a neat, fully labeled and proportioned TS and Ph diagram of the cycle. (10)

Calculate the:

- 3.2 Specific work done by the compressor. (9)
- 3.3 Specific heat transferred in the condenser. (2)
- 3.4 Dryness of the refrigerant at entry to the evaporator. (2)
- 3.5 Specific heat transferred in the evaporator. (1)
- 3.6 Refrigerator's coefficient of performance. (1)

[25]

QUESTION 4

During a Morse test performed on a three cylinder two stroke petrol engine 5,72 kg of fuel is consumed in a 20 minute period while the engine is running at 720 revolutions per minute. The indicated mean effective pressure is 460kPa and the swept volume per cylinder is 0,8 litres. The brake loads when each cylinder is disconnected in turn are:

F₁ = 230 N, F₂ = 228 N, F₃ = 252 N. The brake loads are applied on a brake arm 410mm long. Calculate the:

- 4.1 Brake power per cylinder. (3)
- 4.2 Engine's indicated power. (2)
- 4.3 Engine's mechanical efficiency. (6)
- 4.4 Indicated power per cylinder. (3)
- 4.5 Engine's brake power. (1)
- 4.6 Engine's brake force at full brake power. (1)
- 4.7 Indicated and brake specific fuel consumption in kg/kWh. (4)

[20]

QUESTION 5

Wet steam at a pressure of 860 kPa and a dryness of 0,66 has its specific enthalpy increased at constant pressure by 260 kJ/kg. The steam pressure is then increased adiabatically to a pressure of 1750 kPa. For wet steam $n = 1,13$ and for superheated steam $n = 1,3$.

- 5.1 Calculate the change of specific enthalpy from start to end of the two processes described above.

[15]

TOTAL : 105

ANNEXURE 1: FORMULA SHEET

η_{vol}	$= \frac{V_{ext}}{V_i} = 1 - \frac{V_e}{V_i} \left[\left(\frac{P_2}{P_1} \right)^{\frac{1}{n}} - 1 \right]$	IP	$= IMEP \times L \times A \times N \times E$	MEP	$= \frac{\int W}{V_i}$
$\frac{P_f V_f}{T_f}$	$= \frac{P_1 V_1}{T_1} = \frac{P_2 V_2}{T_2}$	BP	$= 2\pi Ne \times IMEP \times L \times A \times N \times E$	Isotherm WD	$= PV \ln \left(\frac{V_2}{V_1} \right)$
$\int W$	$= \frac{n}{n-1} P_1 V_{ext} \left[\left(\frac{P_2}{P_1} \right)^{\frac{n-1}{n}} - 1 \right]$	T	$= g(M-s) \left(\frac{D+d}{2} \right)$	Polytrop WD	$= \frac{P_1 V_1 - P_2 V_2}{n-1} = \frac{mR(T_1 - T_2)}{n-1}$ heat received - heat rejected
$\int W$	$= IP - BP = m CV$	FP	$= BP / IP$	Iobar $\int W$	$= mc_p [T_3 - T_2] - (T_1 - T_2)$
$\int W$	$= \frac{xn}{n-1} P_1 V_{ext} \left[\left(\frac{P_{ext}}{P_1} \right)^{\frac{n-1}{n}} - 1 \right]$	η_m	$= \eta_{IP}$	Isochor $\int W$	$= mc_v [T_3 - T_2] - (T_1 - T_2)$
K	$= \sqrt{\frac{P_{ext}}{P_1}}$	BMEP	$= IMEP \times \eta_m$	Work Ratio	$= \frac{\int W}{\int W + ve WD}$
COP	$= \frac{\text{refrigerating effect}}{\text{energy input}} = \frac{T_c}{T_h - T_c}$	η	$= \frac{IP}{FP}$	η	$= \frac{\int W}{Q_{rec}} = 1 - \frac{\text{Reject}}{\text{Qrec}}$
h_w	$= h_f + x(h_g - h_f)$	η_{at}	$= \frac{\eta}{\eta_{isent}}$	η_c	$= 1 - \frac{1}{T_c^{1/(n-1)}}$
s_w	$= s_f + x(s_g - s_f)$	$\frac{P_1}{P_2}$	$= \left(\frac{V_2}{V_1} \right)^{\frac{1}{n}}$	η	$= \eta_c \cdot \left[\frac{\alpha \beta^{n-1}}{(\alpha-1)+\sigma(\beta-1)} \right]$
h_{sup}	$= h_g + C_{pg} [T_{sup} - T_{sat}]$	$\frac{T_1}{T_2}$	$= \left(\frac{V_1}{V_2} \right)^{\frac{1}{n-1}}$	Clearance Ratio	$= \frac{V_e}{V_i}$
s_{sup}	$= s_g + C_{pg} \ln \left(\frac{T_{sup}}{T_{sat}} \right)$	$\frac{T_1}{T_2}$	$= \left(\frac{P_1}{P_2} \right)^{\frac{1}{n}}$	η_{isw}	$= \frac{\int W_{iso}}{\int W_{act}}$
h_{sub}	$= h_f - C_{pg} \Delta T $	V	$= x V_s$	H _e	$= M_e \times SHC_e \times AT$
IMEP	$= \left[\frac{\text{diagram area}}{\text{diagram length}} \right] \times Sp.No$	H _c		H _c	$= M_e \times SHC_c \times \Delta T$

<i>P</i>	<i>t₁</i>	<i>V₂</i>	<i>h₁</i>	<i>h₂</i>	<i>f₀</i>	<i>s_i</i>	<i>s_g</i>
kPa	°C	m ³ /kg	kJ/kg	kJ/kg	kJ/kg	kJ/kg/K	kJ/kg/K
12 000	324,7	0,014 26 ¹	1 482	1 197	2 688	3,497	5,600
12 400	327,2	0,013 65	1 508	1 173	2 681	3,523	5,477
12 800	329,6	0,013 05	1 524	1 148	2 672	3,548	5,453
13 200	332,0	0,012 32	1 540	1 122	2 682	3,574	5,429
13 600	334,4	0,011 09	1 556	1 087	2 653	3,599	5,405
14 000	336,8	0,011 60	1 572	1 071	2 643	3,624	5,380
14 400	341,1	0,011 22	1 587	1 044	2 631	3,649	5,356
14 800	343,4	0,010 56	1 603	1 018	2 621	3,674	5,331
15 200	345,2	0,010 12	1 619	980	2 609	3,698	5,305
15 600	345,3	0,009 707	1 625	965	2 598	3,723	5,279
16 000	347,3	0,008 308	1 651	934	2 585	3,747	5,253
16 400	349,3	0,008 925	1 667	906	2 573	3,772	5,227
16 800	351,3	0,008 553	1 683	876	2 559	3,797	5,198
17 200	352,2	0,008 191	1 700	844	2 544	3,824	5,171
17 600	355,1	0,007 838	1 716	812	2 530	3,850	5,143
18 000	357,0	0,007 486	1 735	779	2 514	3,877	5,113
18 400	358,8	0,007 195	1 752	746	2 497	3,903	5,082
18 800	360,6	0,006 839	1 770	710	2 480	3,928	5,050
19 200	362,3	0,006 517	1 786	673	2 461	3,957	5,016
19 600	364,0	0,006 198	1 807	634	2 441	3,985	4,980
20 000	365,7	0,005 977	1 827	592	2 419	4,015	4,941

ρ	t_c	v_g	h_g	h_{fg}	h_u	h_v	s_g	s_v
kPa	°C	m³/kg	kJ/kg	kJ/kg	kJ/kg	kJ/kg	kJ/kg.K	kJ/kg.K
4 100	251.8	0.048 50	1 095	1 705	2 800	2 810	6.058	6.048
4 200	233.7	0.047 51	1 102	1 698	2 800	2 823	6.038	6.048
4 300	254.7	0.048 17	1 109	1 650	2 798	2 836	6.038	6.038
4 400	256.1	0.045 08	1 115	1 683	2 798	2 849	6.029	6.029
4 500	257.4	0.044 04	1 122	1 676	2 798	2 861	6.019	6.019
4 600	258.8	0.043 04	1 129	1 668	2 797	2 874	6.010	6.010
4 700	260.1	0.042 98	1 135	1 661	2 796	2 886	6.000	6.000
4 800	261.4	0.041 16	1 142	1 654	2 795	2 887	5.991	5.991
4 900	262.7	0.040 28	1 148	1 647	2 795	2 899	5.982	5.982
5 000	263.9	0.039 43	1 155	1 640	2 795	2 901	5.974	5.974
5 100	265.2	0.038 61	1 161	1 633	2 794	2 902	5.966	5.966
5 200	266.4	0.037 82	1 167	1 626	2 793	2 945	5.956	5.956
5 300	267.6	0.037 07	1 173	1 619	2 792	2 854	5.946	5.946
5 400	268.8	0.036 33	1 179	1 612	2 791	2 895	5.939	5.939
5 500	269.9	0.035 63	1 185	1 605	2 790	2 976	5.931	5.931
5 600	271.1	0.034 95	1 191	1 598	2 789	2 988	5.923	5.923
5 700	272.2	0.034 29	1 167	1 591	2 788	2 987	5.915	5.915
5 800	273.4	0.033 65	1 202	1 585	2 787	3 007	5.907	5.907
5 900	274.5	0.033 03	1 208	1 578	2 786	3 017	5.898	5.898
6 000	275.6	0.032 44	1 214	1 571	2 785	3 027	5.891	5.891
6 200	277.7	0.031 30	1 225	1 558	2 783	3 047	5.875	5.875
6 400	278.8	0.030 23	1 236	1 545	2 781	3 086	5.860	5.860
6 600	281.6	0.029 22	1 247	1 532	2 779	3 098	5.845	5.845
6 800	283.8	0.028 27	1 257	1 519	2 776	3 104	5.831	5.831
7 000	285.8	0.027 37	1 267	1 506	2 773	3 122	5.818	5.818
7 200	287.9	0.026 52	1 276	1 493	2 771	3 140	5.802	5.802
7 400	289.6	0.025 72	1 286	1 481	2 769	3 157	5.788	5.788
7 600	291.4	0.024 95	1 298	1 468	2 766	3 174	5.774	5.774
7 800	293.2	0.024 22	1 307	1 455	2 762	3 181	5.761	5.761
8 000	295.0	0.023 53	1 317	1 443	2 759	3 208	5.747	5.747
8 200	296.7	0.022 85	1 327	1 430	2 757	3 224	5.734	5.734
8 400	298.4	0.022 23	1 338	1 418	2 754	3 240	5.721	5.721
8 600	300.1	0.021 63	1 345	1 405	2 751	3 258	5.708	5.708
8 800	301.8	0.021 05	1 355	1 393	2 748	3 271	5.695	5.695
9 000	303.5	0.020 50	1 364	1 381	2 745	3 287	5.682	5.682
9 200	304.9	0.019 95	1 373	1 369	2 742	3 302	5.669	5.669
9 400	306.4	0.019 45	1 382	1 356	2 738	3 317	5.657	5.657
9 600	307.9	0.018 97	1 381	1 344	2 735	3 332	5.644	5.644
9 800	309.5	0.018 49	1 389	1 332	2 731	3 346	5.632	5.632
10 000	311.0	0.018 04	1 408	1 320	2 728	3 351	5.616	5.616
10 403	313.4	0.017 18	1 425	1 295	2 720	3 347	5.572	5.572
10 800	315.8	0.016 39	1 442	1 271	2 713	3 347	5.544	5.544
11 200	318.4	0.016 64	1 459	1 247	2 705	3 344	5.524	5.524
11 600	322.1	0.014 94	1 475	1 222	2 697	3 471		